NUMERICAL OPTIMIZATION OF HELICOPTER ROTOR
BLADE DESIGN FOR ACTIVE TWIST CONTROL

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Abstract

The vibration of a helicopter has several different sources, such as the rotor, engine and transmission system. This creates a number of problems with performance, for example poor manoeuvrability, discomfort of the pilot, low fatigue life of the structural components, and, consequently, high operating costs.

At the present time there are in existence various methodologies for vibration reduction, such as Higher Harmonic Control (HHC), Individual Blade Control (IBC), Active Control of Structural Response (ACSR), Active Twist Blade (ATB), and Active Trailing-edge Flap (ATF).

The goal of the presented work is the development of an active twist actuation concept based on the application of Macro Fiber Composite (MFC) actuators. These actuators consist of polyamide films with IDE-electrodes. They are glued on the top and the bottom of piezoceramic ribbons and oriented at ±45° to the spar axis of the rotor blade. The interdigitated electrodes deliver the electric field required to activate the piezoelectric effect in the fibers and this, in turn, creates a stronger longitudinal piezoelectric effect along the length of the fibers. The properties and orientation of piezoelectric actuators cause the MFC actuators to induce shear stresses and thus distribute a twisting movement along the blade.

The design methodology, based on the planning of experiments and response surface technique, has been developed for an optimum placement of Macro Fiber Composite (MFC) actuators in helicopter rotor blades. The optimisation problem for the optimum placement of actuators in helicopter rotor blades has been formulated on the results of parametric study. The investigated helicopter has a rotor blade which is equipped with NACA23012 airfoil and has a rectangular shape with an active part radius of 1.56 m and a chord length of 0.121 m. The blade also consists of D-spar. This is made of unidirectional fiberglass, reinforced plastic (UD GFRP), a skin made of ±45° GFRP, a foam core, balancing weight and MFC actuators placed on the top and bottom of the skin. The 3D finite element model of the rotor blade has been built by ANSYS. The rotor blade skins are modelled with linear layered structural shell elements (SHELL99), balancing weights and foam with 3D 20-node structural solid elements (SOLID186). The spar is modelled with shell and solid elements.

The optimisation results have been obtained for design solutions, connected with an application of active materials, and checked by the finite element calculations.
1. Introduction

The vibration of a helicopter has several different sources, such as the rotor, engine and transmission system. This creates a number of problems with performance, for example poor manoeuvrability, discomfort of the pilot, low fatigue life of the structural components, and, consequently, high operating costs.

Traditional helicopter vibration control methods are based on the application of Higher Harmonic Control (HHC) and Individual Blade Control (IBC). However, because of the recent development of smart materials, a new active control technique has begun to be implemented for the control of rotor blade vibration, noise reduction and enhanced performance.

Recently, several active control approaches have been investigated. These have used smart material actuation to achieve individual blade control without the need for complex mechanisms in the rotating frame. The application of such materials for control has resulted in the production of two innovative designs: the trailing edge flap rotor and active twist rotor.

Early studies on the concept of an active twist rotor were done by Chen and Chopra in 1996 [1, 2]. According to their designs, the rotor blade incorporated discrete dual-layer monolithic piezopatch elements embedded at +45° under the upper skin and -45° under the lower skin of the rotor blade. Experiments were conducted on a two bladed, hingeless rotor with a 6 foot diameter and built to a 1/8 Froude scale. Different piezoceramic arrangements were analyzed along with different rotation speeds. The maximum tip twists at resonance frequencies were 0.35° and 1.1°.

Later, in 1998, Bernhard and Chopra [3, 4] developed an active, on blade vibration-reduction system which used smart, active blade tips (SABT) and was driven by a piezo-induced, bending–torsion coupled actuator. The beam was designed specifically to fit within the rotor blade profile and was used essentially as a pure torsional actuator. The beam lay-up was selected so that if the inboard end was clamped, the outboard end resulted in a twist. This twist was measured by the deflection of the SABT. A Mach-scale active twist rotor was fabricated and tested using the SABT concept. The rotating twist amplitudes with SABT are similar to the tip twist amplitudes reported by Chen and Chopra [8, 9].

In 1998 a new approach to smart materials was adopted by Rodgers and Hagood [5, 6]. They attempted to develop an integral, twist-actuated rotor which would allow for individual blade control. Active fiber composites (AFC) were integrated within a composite rotor blade to induce a twisting movement. These were active plies oriented at 45° angles to the blade span in order to induce shear stresses and the distribution of a twisting movement along the blade. A CH-47D blade was designed to a 1/6 Mach-scale and was then fabricated and tested in hover at the MIT hover test facility. Good correlation was obtained between theory and the results of experiments.

The goal of the present work is the development of an active twist actuation concept based on the application of Macro Fiber Composite (MFC) actuators [7, 8]. A Macro Fiber Composite was developed at NASA Langley Research Center. The actuators consist of polyamide films with interdigitated electrodes that are glued on the top and bottom of piezoceramic fibres (Fig. 1).
analysis with a thermal load is carried out to characterize an active twist of the helicopter rotor blade. Design solutions for an application of active materials are studied to estimate the effectiveness.

2. Helicopter rotor blade

An investigated helicopter rotor blade (Fig. 3) is equipped with NACA23012 airfoil and has a rectangular shape with active part radius $R = 1.56$ m and chord length $c = 0.121$ m.

![Fig. 3. Active part of rotor blade with MFC.](image)

This rotor blade consists of D-spar made of unidirectional GFRP (Fiberglass Reinforced Polymer), skin made of $\pm 45^\circ/\pm 45^\circ$ GFRP, a foam core, MFC actuators and balancing weight (Fig. 4). The thickness of the GFRP skin layer is 0.125 mm whilst the thickness of MFC layer is 0.3 mm. The material properties of the rotor blade components are as follows:

- **GFRP**: $E_x = 11.981$ GPa, $E_y = 11.981$ GPa, $E_z = 45.166$ GPa, $G_{xz} = 4.583$ GPa, $G_{yz} = 4.583$ GPa, $G_{xy} = 1.289$ GPa, $\nu_{xz} = 0.238$, $\nu_{yz} = 0.238$, $\nu_{xy} = 0.325$, $\rho = 2008$ kg/m$^3$
- **Foam (Rohacell 51 FX)**: $E = 0.035$ GPa, $G = 0.014$ GPa, $\nu = 0.25$, $\rho = 52$ kg/m$^3$
- **Lead**: $E = 13.790$ GPa, $G = 2.000$ GPa, $\nu = 0.44$, $\rho = 11300$ kg/m$^3$
- **MFC**: $E_x = 30.0$ GPa, $E_y = 15.5$ GPa, $E_z = 15.5$ GPa, $G_{xz} = 10.7$ GPa, $G_{yz} = 10.7$ GPa, $G_{xy} = 5.7$ GPa, $\nu_{xz} = 0.4$, $\nu_{yz} = 0.4$, $\nu_{xy} = 0.35$, $d_{31} = 4.18 \times 10^{-10}$ m/V, $d_{32} = -1.98 \times 10^{-10}$ m/V, $\rho = 4700$ kg/m$^3$

![Fig. 4. Cross-section of the helicopter rotor blade.](image)

3. Finite element analysis

The 3D finite element model of the rotor blade is produced by ANSYS (Fig. 5). The rotor blade skin and spar “moustaches” are modelled with linear layered, structural shell elements (SHELL99). The spar and foam, however, are produced using 3D 20 node structural solid elements (SOLID186). The clamped boundary conditions are applied from one end-side of the rotor blade.

![Fig. 5. Finite element model of the helicopter rotor blade without foam.](image)

The thermal analogy based on the analogy between the piezoelectric effect and thermoelastic effect. An applied electric field is modelled as a thermal load. The correspondence between piezoelectric strain and thermal strain is obtained by taking:

$$\alpha_{ij} = \frac{d_{ij}}{\Delta_{ES}}$$

where $d_{ij}$ is the effective piezoelectric constant and $\Delta_{ES}$ is the electrode spacing taken as $\Delta_{ES}=0.5$ mm.

Then, steady-state thermal analysis is carried out to determine a torsion angle of the rotor blade (Fig. 6), static torsion analysis is employed to determine a location of the elastic axis and modal analysis is used to determine the first torsion eigenfrequency of the rotor blade.

![Fig. 6. Twist of the helicopter rotor blade.](image)

4. Parametric study

Before the formulation of the optimisation problem, the parametric study has been carried out with the purpose to decrease a number of design parameters (Fig. 7). By doing this the accuracy of the obtained optimal results will be increased. The findings of parametric study have been investigated in previous work [10].
The skin thickness is examined as a discrete value design parameter with the step $\Delta t_{(+45/-45)} = 0.25$ mm. The smallest influence has been shown by spar “moustaches” length, web length and web thickness. The influence on the torsional angle of the spar “moustaches” thickness and web thickness together is better than only the spar “moustaches” thickness. The voltage has influence on the strain. But this influence is less than permissible. For this reason, web thickness, web length, voltage and spar “moustaches” thickness have been excluded from the set of design parameters used in this optimisation.

5. Optimal design

Due to the large dimensions of the numerical problem to be solved, the non-direct optimisation technique should be applied. Indeed, the application of direct minimisation algorithms and multiple finite element analysis is too expensive from a computational point of view. For this reason, an optimisation methodology is developed employing the method of experimental design and response surface technique.

5.1 Formulation of optimisation problem

An optimisation problem for the optimum placement of actuators in helicopter rotor blades (Fig. 8) has been formulated based on the results of parametric study [10] and taking into account the producer’s requirements:

Design parameters: $\{x\} = \{l, t_{\text{skin}}, t_{\text{spar}}, L\}$

- $16 \leq l \leq 24$ mm
- $0.25 \leq t_{\text{skin}} \leq 1.25$ mm
- $0.50 \leq t_{\text{spar}} \leq 2.50$ mm
- $16 \leq L \leq 100$ mm

Constraints:

- $22 \leq y_{cg} \leq 30$
- $10 \leq y_{ea} \leq 25$
- $m \leq 1.35$
- $f_{T1} \geq 59.15$

where $l$ - spar circular fitting (mm), $t_{\text{skin}}$ - skin thickness (mm), $t_{\text{spar}}$ - spar thickness and web thickness together (mm), $L$ - MFC chord-wise length (mm), $y_{cg}$ - location of the centre of gravity (%), $y_{ea}$ - location of the elastic axis (%), $m$ - mass of cross-section (kg/m), $f_{T1}$ - first torsion eigenfrequency (Hz). The length of spar is 56 mm and Voltage is 1000 V.

Later studies of parametric research for two different applications of the active material (Fig. 9) have been studied to determine their effectiveness. As can be seen from the graphs (Fig. 10), variant 2 is considerably more effective than variant 1 in relation to active twist of the helicopter rotor blade. For this reason, the second variant of application of smart material has been chosen as a solution to the optimization problem, when active materials apply on the top and bottom of skin.

5.2. Experimental design and FEM simulations

The program EDAOpt [11] was used for the generation of the experimental design. D-optimal experimental design is formulated for 4 design parameters and 30 experiments. Then, in the points of the plan of experiments, both the structural static analysis with a thermal load and modal analysis are carried out to determine the torsion angles in the rotor blade and the first torsion eigenfrequency respectively. As with additional parameters, a location of the centre of gravity and rotor blade mass are found from the finite element model. Determining the elastic axis location is a more complicated problem and requires a solution to the additional static torsion...
problem. This involves the application of two forces independently from both sides of the sought elastic centre [12].

5.3. Response surfaces.

At the time of the present study a form of the equation of regression was previously unknown [13]. There are two requirements for the equation of regression: accuracy and reliability. Accuracy is characterised as a minimum of standard deviation of the table data from the values given by the equation of regression. By increasing the number of terms in the equation it is possible to obtain a complete agreement between the table data and the values given. However, it is necessary to note that prediction at the intervals between the table points can be poor. For an improvement of prediction, it is necessary to decrease the distance between the points of experiments by increasing the number of experiments or by decreasing the domain of factors. Reliability of the equation of regression can be characterised by an affirmation that standard deviations for the table points and for any other points are approximately the same. Obviously the reliability is greater for a smaller number of terms of the equation of regression.

The equation of regression can be written in the following form:

\[ y = \sum_{i=1}^{p} A_i f_i(x_j) \]

where \( A_i \) are the coefficients of the equation of regression, \( f_i(x_j) \) are the functions from the bank of simple functions \( \theta_1, \theta_2, \ldots, \theta_m \) which are assumed as,

\[ \theta_m(x_j) = \prod_{i=1}^{s} x_j^{\zeta_{mi}} \]

where \( \zeta_{mi} \) is a positive or negative integer including zero. Synthesis of the equation from the bank of simple functions is carried out in two stages: selection of perspective functions from the bank and then step-by-step elimination of the selected functions.

At the first stage all variants are tested with the least square method and the function which leads to the minimum of the sum of deviations is chosen for each variant. On the second stage, the elimination is carried out using the standard deviation

\[ \sigma_0 = \sqrt{\frac{S}{k-p+1}} \]

or correlation coefficient

\[ c = \left(1 - \frac{\sigma}{\sigma_0}\right) \times 100\% \]

where \( k \) is the number of experimental points, \( p \) is the number of selected perspective functions and \( S \) is the minimum sum of deviations. It is more convenient to characterise the accuracy of the equation of regression by the correlation coefficient (Figure 11).

If insignificant functions are eliminated from the equation of regression, a reduction of the correlation coefficient is negligible. If in the equation of regression only significant functions are presented, elimination of one of them leads to an important decrease in the correlation coefficient.

Response surfaces for all behaviour functions have been obtained with the correlation coefficients around 90% and higher. They have been verified by the finite element solutions in the points different from the points taken in the plan of experiments. Results of verification are presented in Figures 12-14.
Fig 13. Accordance between approximation function and control points for elastic axis and spar circular fitting.

$\ell = 20 \text{ mm}$, $t_{\text{skin}} = 0.75 \text{ mm}$, $t_{\text{spar}} = 1.5 \text{ mm}$

**Table 1.** Design parameters

<table>
<thead>
<tr>
<th>Design parameters</th>
<th>$\ell$ mm</th>
<th>$t_{\text{skin}}$ mm</th>
<th>$t_{\text{spar}}$ mm</th>
<th>$L$ mm</th>
</tr>
</thead>
<tbody>
<tr>
<td>Response surfaces</td>
<td>22</td>
<td>0.25</td>
<td>0.50</td>
<td>82</td>
</tr>
<tr>
<td>FEM</td>
<td>22</td>
<td>0.25</td>
<td>0.50</td>
<td>82</td>
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</table>

**Table 2.** Constraints

<table>
<thead>
<tr>
<th>Constraints</th>
<th>$y_{cg}$ %</th>
<th>$y_{ea}$ %</th>
<th>$m$ kg/m</th>
<th>$f_{T1}$ Hz</th>
</tr>
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<tbody>
<tr>
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<td>29.4</td>
<td>18.1</td>
<td>1.20</td>
<td>59.77</td>
</tr>
<tr>
<td>FEM</td>
<td>29.0</td>
<td>17.7</td>
<td>1.23</td>
<td>58.88</td>
</tr>
<tr>
<td>$\Delta$ %</td>
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<td>2.3</td>
<td>2.5</td>
<td>1.5</td>
</tr>
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</table>

**Table 3.** Objective function and effectiveness

<table>
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<th>Objective function</th>
<th>Effectiveness</th>
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</thead>
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<tr>
<td>$\phi$, $^\circ$</td>
<td>$\phi/R$, $^\circ$/m</td>
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<tr>
<td>Response surfaces</td>
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<tr>
<td>FEM</td>
<td>5.39</td>
</tr>
<tr>
<td>$\Delta$ %</td>
<td>0.0</td>
</tr>
</tbody>
</table>

5.4. Optimisation results

The non-linear, global optimisation problem is executed by the Improved Multistart Random Search [14] using the response surfaces obtained. In the first stage, the design problem has been solved with the purpose of obtaining a maximum torsion angle. The optimisation is given in Tables 1-3 together with the finite element solutions obtained in the optimal point. As can be seen in the tables, the difference between finite element and response surface solutions is reasonable.

5.5. Parametric study

After optimisation, additional parametric study has been carried out for designer convenience and to investigate the influence of different design parameters on behavioural functions. The dependencies of the torsion angle on MFC chord-wise length and spar circular fitting, first torsion frequency on MFC chord-wise length, and skin thickness are given in Figures 15-17.

**Fig 14.** Accordance between approximation function and control points for frequency and MFC chord-wise length.

**Fig 15.** The dependencies of torsion angle on MFC chord-wise length and spar circular fitting.

**Fig 16.** The dependencies of first torsion frequency on MFC chord-wise length and skin thickness.
6. Conclusions

An optimisation problem for the optimum placement of actuators in helicopter rotor blades has been formulated on the results of parametric study using the finite element method.

The methodology, based on the planning of experiments, the response surface technique, and parametric study, has been developed for the optimum placement of actuators in helicopter rotor blades. To describe the behaviour of a twisted rotor blade, the finite element method has been applied in the sample points of experimental design. For this purpose the structural static analysis with a thermal load, using a 3D finite element model, has been developed by ANSYS. Minimisation problems have been solved by the Improved Multistart Random Search, using the approximating functions instead of the original functions.

From the optimisation results of the present study, and in comparison with previous studies, it was observed that the 2nd variant arrangement of MFC is more effective than the 1st in the production of a maximum torsion angle under voltage on 26%.

From the approximation function and using the obtained optimal results there can be found a compromise between the necessary solutions.

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References